# On a Sandia Structural Mechanics Challenge Problem 

Jan Chleboun<br>chleb@math.cas.cz<br>Mathematical Institute, Academy of Sciences<br>and<br>Faculty of Mechanical Engineering, Czech Technical University<br>Prague, Czech Republic

PANM 13, Prague
May 28-31, 2006

## Prediction problem:

Does the displacement $\delta_{P}$ exceed a given limit ( 3 mm )?


We know:

- the geometry
- the load
- the mathematical model

We have limited information about $E$, the modulus of elasticity.


Information provided about $E$ :

- five local values $E_{0}$
- five averaged values $E_{20}$ inferred from the elongation of sample rods 20 cm long
- two averaged values $E_{80}$ inferred from the elongation of sample rods 80 cm long
- $\delta_{Q}$, a displacement of a "similar" structure nicknamed the accreditation problem


The prediction problem was proposed by Ivo Babuška, Fabio Nobile, and Raul Tempone as one of the uncertain input data problems to challenge the participants of Validation Challenge Workshop, Sandia National Laboratories, Albuquerque, NM, USA, May 21-23, 2006.
In the problem, three levels of information (i.e., sets of measurements) are offered. We use the poorest set.

## Difficulties:

- insufficient number of experiments
- uncertain probability distribution
- poor estimates
of probability-related parameters



## Stochastic process

$E \equiv E(x, \omega)$ is a stationary random field (w.r.t. $x$ ); $x \in\left[0, L_{i}\right]$ where $L_{i}$ is the length of the $i$-th rod. For some purposes, $E(x, \omega)$ can be reduced to $E_{0}(\omega)$, the field independent of $x$.
The expected value of $E$ (the mean): $E_{\mathrm{m}}=\mathbb{E}\left(E_{0}(\omega)\right)=$ constant independent of $x$ and $i$.
We have to assume that $E(x, \omega)$ and $E(y, \omega)$ are not independent especially if $x$ is "close" to $y$.

## Idea:

- Choose intervals $I_{0}$ and $I_{20}$ such that they contain the measured values of $E_{0}$ and $E_{20}$, respectively.
- Assume a probability distribution of $E_{0}$.
- For $1 / E$, assume a covariance function with an unknown correlation length $L_{\text {corr }}$.
- Calculate the correlation length $L_{\text {corr }}$.
- By knowing $L_{\text {corr }}$, infer an interval for $E_{80}$ and check it against the measured values of $E_{80}$.
- Infer an interval for $\delta_{Q}$ and check it against the value of $\delta_{Q}$ coming from the accreditation test.
- Infer an interval for $\delta_{P}$ and check it against the 3 mm limit given in the prediction problem. Try to make a conclusion.

Much space for expert opinion!
Remark: B.\&N.\&T. call the measured $E_{20}$ values calibration data and the measured $E_{80}$ values validation data.


The intervals are constructed around a central value (dashed) and interpreted as either the respective intervals in which both $E_{0}$ and $E_{20}$ are uniformly distributed or the intervals covering 95\% of normally distributed values $E_{0}$ and $E_{20}$.

Let us recall that $E_{0}(\omega)$ is a random field of local values of $E$, that is, a field identical to $E(x, \omega)$ except for the localization at a particular $x$.

We assume

$$
\begin{aligned}
\operatorname{cov}\left(\frac{1}{E}\right) & =\mathbb{E}\left[\frac{1}{E(x, \omega)} \frac{1}{E(y, \omega)}\right]-\left(\mathbb{E}\left[\frac{1}{E_{0}(\omega)}\right]\right)^{2} \\
& =\operatorname{var}\left(\frac{1}{E_{0}}\right) g\left(x, y, L_{\mathrm{corr}}\right)
\end{aligned}
$$

where

$$
g\left(x, y, L_{\text {corr }}\right)=\exp \left(-\frac{|x-y|}{L_{\text {corr }}}\right)
$$

Other choices of $g$ are possible. Take $g\left(x, y, L_{\text {corr }}\right)=\exp \left(-\frac{|x-y|^{2}}{L_{\text {corr }}^{2}}\right)$, for instance.

Equation for $L_{\text {corr }}$
If a uniform rod of length $L$ and cross section area $A$ is axially loaded by a force $F$, then $\delta_{L}$, its elongation, is a random variable:

$$
\delta_{L}(\omega)=\frac{F}{A} \int_{0}^{L} \frac{1}{E(x, \omega)} d x
$$

Then

$$
\begin{aligned}
\operatorname{var}\left(\delta_{L}\right) & =\mathbb{E}\left[\delta_{L}^{2}\right]-\left(\mathbb{E}\left[\delta_{L}\right]\right)^{2}=\ldots \text { after some algebra } \ldots \\
& =\frac{F^{2}}{A^{2}} \int_{0}^{L} \int_{0}^{L} \mathbb{E}\left[\frac{1}{E(x, \omega)} \frac{1}{E(y, \omega)}\right]-\left(\mathbb{E}\left[\frac{1}{E_{0}(\omega)}\right]\right)^{2} d x d y \\
& =\frac{F^{2}}{A^{2}} \int_{0}^{L} \int_{0}^{L} \operatorname{cov}\left(\frac{1}{E}\right) d x d y \\
& =\frac{F^{2}}{A^{2}} \operatorname{var}\left(\frac{1}{E_{0}}\right) \int_{0}^{L} \int_{0}^{L} \exp \left(-\frac{|x-y|}{L_{\mathrm{corr}}}\right) d x d y
\end{aligned}
$$

However, if we define $E_{L}$ as the effective modulus of elasticity inferred from the prolongation of the rod of length $L$, we obtain

$$
\begin{aligned}
\delta_{L}(\omega) & =\frac{F L}{A} \frac{1}{E_{L}(\omega)} \\
\operatorname{var}\left(\delta_{L}\right) & =\frac{F^{2} L^{2}}{A^{2}} \operatorname{var}\left(\frac{1}{E_{L}}\right) .
\end{aligned}
$$

By comparing both equations, we eliminate $\operatorname{var}\left(\delta_{L}\right)$ and arrive at

$$
\begin{equation*}
\frac{\operatorname{var}\left(1 / E_{L}\right)}{\operatorname{var}\left(1 / E_{0}\right)}=\frac{1}{L^{2}} \int_{0}^{2} \int_{0}^{2} \exp \left(-\frac{|x-y|}{L_{\text {corr }}}\right) d x d y \tag{1}
\end{equation*}
$$

To solve (1), we evaluate $\operatorname{var}\left(1 / E_{0}\right)$ by means of the assumed probability distribution of $E_{0}$ in the interval $I_{0}$. We evaluate $\operatorname{var}\left(1 / E_{L}\right)$, where $L=20 \mathrm{~cm}$, in a similar way using $I_{20}$. After exact integration of the r.h.s. of (1) (done by Maple), the r.h.s. becomes a function of $L_{\text {corr }}$, and (1) can be solved numerically for $L_{\text {corr }}$.

As soon as $\operatorname{var}\left(1 / E_{0}\right)$ is fixed by assumption and $L_{\text {corr }}$ is known from (1), we can use (1) to directly calculate $\operatorname{var}\left(1 / E_{L}\right)$ for $L=80 \mathrm{~cm}$ and other lengths. We assume that $\operatorname{var}\left(1 / E_{80}\right)$ corresponds to either a uniform or normal distribution of $E_{80}$. Under these assumptions, we can infer $I_{80}$ and check whether or not the validation data lie in $I_{80}$.

In a similar way but with much less effort, we can infer

$$
\mathbb{E}\left[\delta_{L}\right]=\frac{F L}{A} \mathbb{E}\left[\frac{1}{E_{0}}\right]
$$

Since $\delta_{Q}$, see the accreditation problem, can be expressed as a linear combination of $\delta_{L_{i}}, i=1,2,3,4$, the same technique enables us to obtain the mean value of $\delta_{Q}$ and $\operatorname{var}\left(\delta_{Q}\right)$.


Similarly, we infer the mean value of $\delta_{P}$ and $\operatorname{var}\left(\delta_{P}\right)$, the quantities important for addressing the prediction problem.


Remark: The bending of transversaly loaded beams is expressed through the Green function. To compute the corresponding variation of the vertical displacements, integrals such as

$$
\begin{equation*}
\int_{0}^{L} \int_{0}^{L} \phi(x) \psi(y) \exp \left(-\frac{|x-y|}{L_{\mathrm{corr}}}\right) d x d y \tag{2}
\end{equation*}
$$

have to be evaluated. In (2), the product $\phi \psi$ is a continuous piecewise quadratic or cubic function. Again, Maple is able to analytically integrate expression (2) and convert the resulting formulae into Matlab code.

To get an insight, it also helps to apply the Monte Carlo method to both structures, though it simulates partly different mathematical model. Indeed, sample structures are generated with effective elasticity moduli $\left(E_{L_{i}}\right)$ that are used even in the beam loaded by the transversal force; this is not exactly the model that we have studied.


The results of our calculations will be presented in graphs that, we believe, can help the analyst to get some insight into the response of the prediction problem to the assumptions made.

E-mean coef. $=1 \quad$ E-local range coef. $=0.33 \quad \mathrm{E}_{0}-\mathrm{E}_{20}$ ratio $=0.25$



E-mean coef. $=1.01 \quad$ E-local range coef. $=0.36 \quad \mathrm{E}_{0}-\mathrm{E}_{20}$ ratio $=0.3$



E-mean coef. $=0.99 \quad$ E-local range coef. $=0.30 \quad E_{0}-E_{20}$ ratio $=0.3$





E-mean coef. $=0.98 \quad$ E-local range coef. $=0.27 \quad \mathrm{E}_{0}-\mathrm{E}_{20}$ ratio $=0.3$



E-mean coef. $=0.97 \quad$ E-local range coef. $=0.24 \quad \mathrm{E}_{0}-\mathrm{E}_{20}$ ratio $=0.35$



## Observations

- The most important quantity is $E_{\mathrm{m}}=\mathbb{E}\left[E_{0}\right]$. The smaller $E_{\mathrm{m}}$, the greater probability that $\delta_{P}$ exceeds the limit.
- If $E_{\mathrm{m}}$ becomes too small, say less than $0.98 E_{\mathrm{M}}$, where $E_{\mathrm{M}}$ is the mean of measured $E$, then it is harder to comply with the validation and accreditation data.
- The greater $E_{\mathrm{m}}$ (above $E_{M}$ ), the smaller $\delta_{\mathrm{P}}$ (good news). However, it is harder to comply with the calibration tests. Moreover the calibration dataset then becomes more and more "one-sided", which is less and less probable.
- Although, at the first glance, the predictions based on $E$ uniformly distributed seems to be worse (closer to 3 mm ) than the predictions based on the Gaussian distribution of $E$, they are not that much different because the uniform distribution leads to short "tails", see the histograms of $\delta_{P}$.

Answers to dilemma Yes $\left(\delta_{P} \geq 3\right) \quad$ No $\left(\delta_{P}<3\right)$

1) If you follow the rule that a model both strict and fitting to the data is the best choice, say No. You will sleep like an innocent baby. 2) If you are a realist but if you believe that things mostly end in good, say No. You will sleep well.
2) If you know the harsh side of life, say No. You will feel that you have more than a fifty-fifty chance to be right.
3) If it is a matter of life and death, say No. Simply try to believe in my sixth sense.
4) If you do not feel any inclination to yes or no and if you do not hear an inner voice, quit problems with uncertain data.

I thank Ivo Babuška, Fabio Nobile, and Raul Tempone for inventing this puzzle and for giving me lectures on variance, covariance, and loaded beams.

